




C.A.R.E. PHYS 213 Quiz 2
Review Session



CARE / CARE PHYS 213 Exam Review Session

 No location specified

 Enable notifications



Open Queue

Delete All Questions

Show staff message

This affects anyone using this queue



On-Duty Staff

No on-duty staff

Join

Queue

Welcome

Tutorials

Tuesday

Workshop

Slides

Solutions

Also, here

install

Jupyter

Good luck



session!

during these times:

in the test. If you do not have a Jupyter Notebook environment, here is a suggestion for this coding example!

This queue is closed. Check back later!

Units for the Exam

- Kinetic Theory of Ideal Gases
- Quasistatic Processes
- Thermodynamic Cycles
- Gibbs Free Energy

Ideal gas and Equipartition

- Ideal Gas: Approximation of particles as points with no interactions:

- Follows **ideal gas law**: $pV = NkT = nRT$

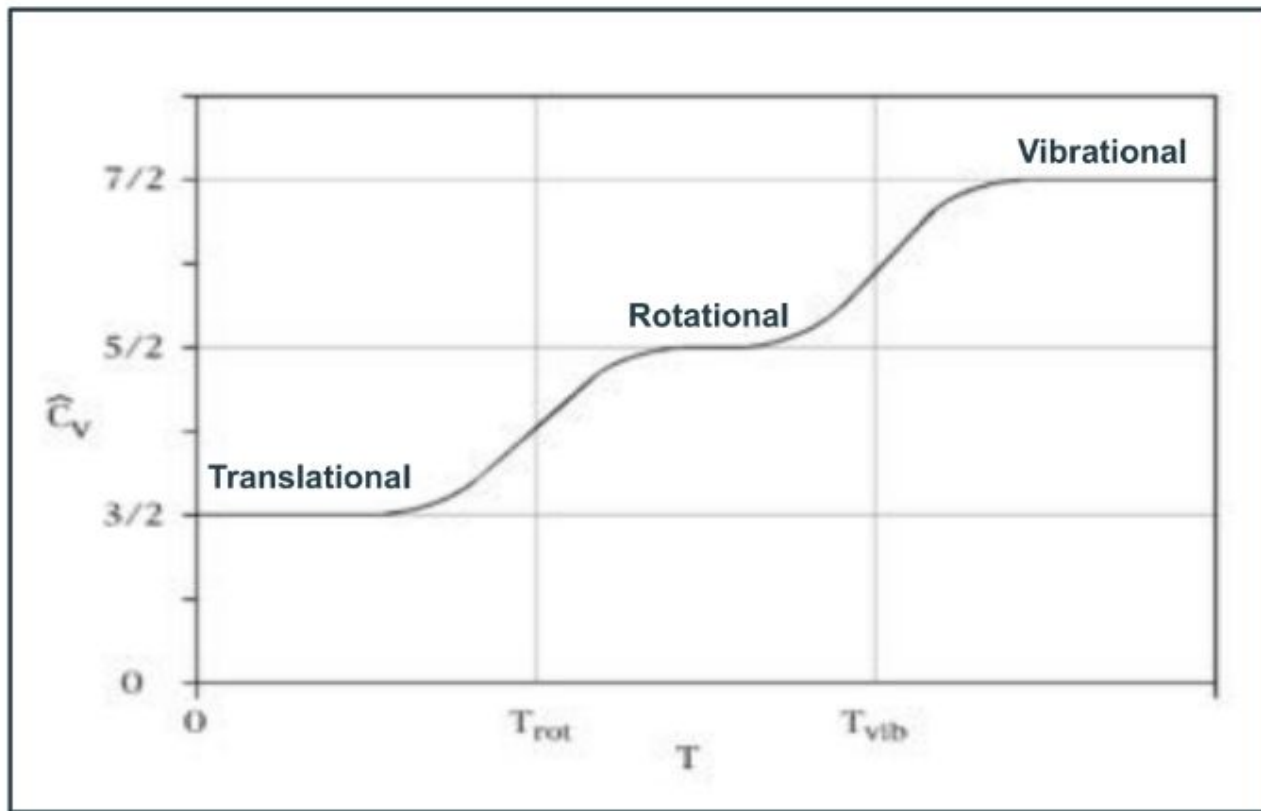
- **Equipartition**: each degree of freedom contributes $\frac{1}{2} kT$ of energy

- Internal energy in each particle:

- $U = (N_{DOF}/2)kT$

- Molar heat capacity:

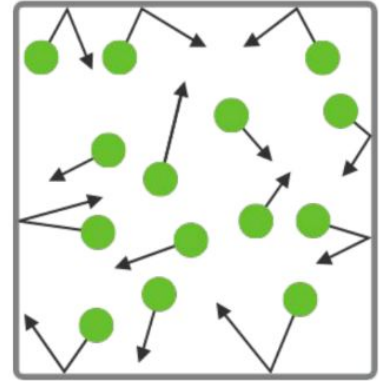
- $c_M = (N_{DOF}/2)kN_A$



Root-Mean-Square Velocity

- v_{rms} : Average (**translational**) velocity of gas particles
- Translational Kinetic Energy: $KE_{translational} = 1/2 m (v_{rms})^2$
- Relationship to temperature:

$$\frac{1}{2} m v_{rms}^2 = \frac{3}{2} kT$$



- This only applies to **ONE PARTICLE**
- **Notice:** this does **NOT** depend on the number of DOFs; it's **ALWAYS** $(3/2)kT$
 - Why? Translational KE only depends on the translational modes motion (there are only 3 translational modes: v_x, v_y, v_z)

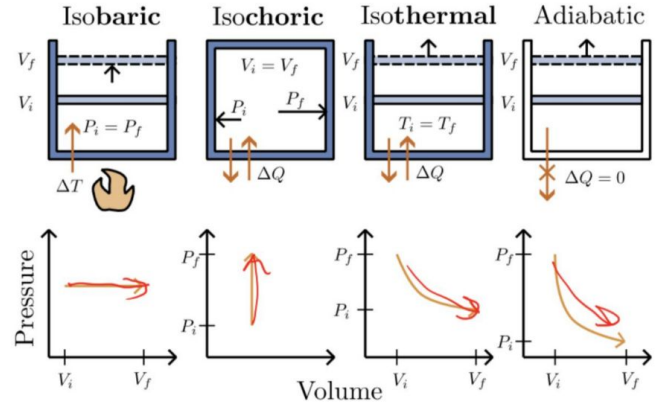
Thermodynamic Processes

- Isochoric or Isovolumetric
 - Constant VOLUME

- Isobaric
 - Constant PRESSURE

- Isothermal
 - Constant TEMPERATURE, REVERSIBLE

- Adiabatic
 - Constant HEAT ($dQ = 0$), REVERSIBLE



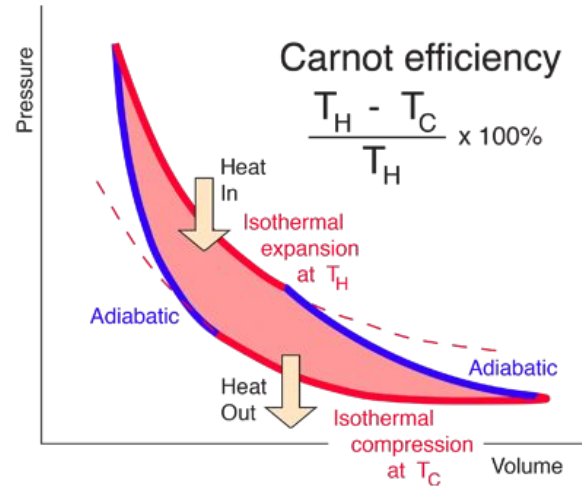
$$(\Delta S_{total} = 0, \Delta U = 0)$$

$$(\Delta S_{total} = 0, \Delta Q = 0)$$

Reversible Processes

- Isothermal + Adiabatic processes are REVERSIBLE

- $\Delta S_{total} = 0$ (no change in entropy)
- For isothermal processes:
 - $PV = \text{constant}$
- For adiabatic processes:
 - $PV^\gamma = \text{constant}$
 - $\gamma = (2/N_{DOF}) + 1$ (given on equation sheet)



Example Problem - Adiabatic Process

- Assume we have a gas undergoing an adiabatic process, determine the work done given the following parameters:
 - $V_i = 10 \text{ m}^3$, $p_i = 10 \text{ kPa}$
 - $V_f = 4 \text{ m}^3$, $N_{\text{DOF}} = 3$

1. Calculate γ

2. Find pV^γ

3. Calculate W

4. Calculate W

Example Problem - Adiabatic Process

- Assume we have a gas undergoing an adiabatic process, determine the work done given the following parameters:
 - $V_i = 10 \text{ m}^3$, $p_i = 10 \text{ kPa}$
 - $V_f = 4 \text{ m}^3$, $N_{\text{DOF}} = 3$

$$\gamma = \frac{2}{N_{\text{DOF}}} + 1 = \frac{2}{3} + 1 = \frac{5}{3}$$

2. Find pV^γ

3. Calculate W

4. Calculate W

Example Problem - Adiabatic Process

- Assume we have a gas undergoing an adiabatic process, determine the work done given the following parameters:

- $V_i = 10 \text{ m}^3$, $p_i = 10 \text{ kPa}$
- $V_f = 4 \text{ m}^3$, $N_{\text{DOF}} = 3$

$$\gamma = \frac{2}{N_{\text{DOF}}} + 1 = \frac{2}{3} + 1 = \frac{5}{3}$$

$$pV^\gamma = \text{Constant} = p_i V_i^\gamma = (1000)(10)^{\frac{5}{3}} = 464158.88$$

3. Calculate W

4. Calculate W

Example Problem - Adiabatic Process

- Assume we have a gas undergoing an adiabatic process, determine the work done given the following parameters:

- $V_i = 10 \text{ m}^3$, $p_i = 10 \text{ kPa}$
- $V_f = 4 \text{ m}^3$, $N_{\text{DOF}} = 3$

$$\gamma = \frac{2}{N_{\text{DOF}}} + 1 = \frac{2}{3} + 1 = \frac{5}{3}$$

$$pV^\gamma = \text{Constant} = p_i V_i^\gamma = (1000)(10)^{\frac{5}{3}} = 464158.88$$

$$W = \int_{V_i}^{V_f} p dV = \int_{V_i}^{V_f} \frac{C}{V_i^\gamma} dV = \left[C \frac{V_f^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right] - \left[C \frac{V_i^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right]$$

4. Calculate W

Example Problem - Adiabatic Process

- Assume we have a gas undergoing an adiabatic process, determine the work done given the following parameters:

- $V_i = 10 \text{ m}^3$, $p_i = 10 \text{ kPa}$
- $V_f = 4 \text{ m}^3$, $N_{\text{DOF}} = 3$

$$\gamma = \frac{2}{N_{\text{DOF}}} + 1 = \frac{2}{3} + 1 = \frac{5}{3}$$

$$pV^\gamma = \text{Constant} = p_i V_i^\gamma = (1000)(10)^{\frac{5}{3}} = 464158.88$$

$$W = \int_{V_i}^{V_f} p dV = \int_{V_i}^{V_f} \frac{C}{V_i^\gamma} dV = \left[C \frac{V_f^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right] - \left[C \frac{V_i^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right]$$

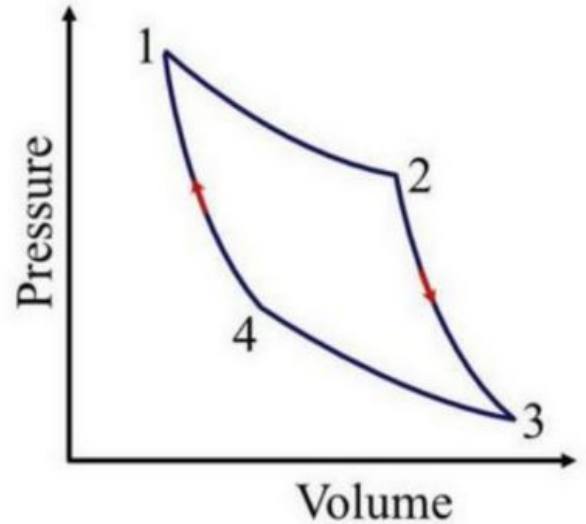
$$W = \left[(464158.88) \frac{4^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right] - \left[(464158.88) \frac{10^{(-\frac{5}{3}+1)}}{-\frac{5}{3}+1} \right] \approx -126.30 \text{ kJ}$$

$$W \approx -126.30 \text{ kJ}$$

p-V Diagrams

- Used to visualize thermodynamic cycles
- Area enclosed in the curve is equal to the work per cycle
 - **Clockwise direction: work is positive** (engine did work)
 - **Counterclockwise direction: work is negative** (work done on engine)

$$W_{\text{by}} = \int_{V_i}^{V_f} p \, dV$$



Heat Engines

- Cycles of Thermodynamic processes are used to make **engines**, **heat pumps**, and **refrigerators**

- **Efficiency of engines:**

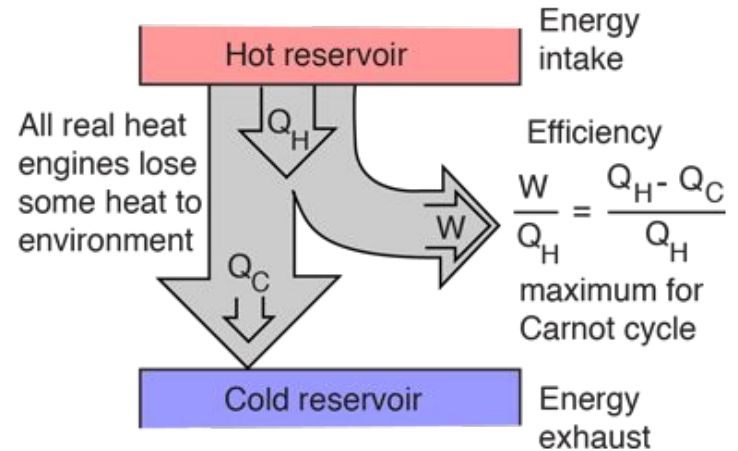
$$\varepsilon = \frac{W_{by}}{Q_H} \leq 1 - \frac{T_C}{T_H}$$

- **COP of pumps and refrigerators:**

- **Heat pump:** $\text{COP} = \frac{Q_H}{W_{on}} \leq \frac{1}{1 - \frac{T_C}{T_H}}$

- **Refrigerator:** $\text{COP} = \frac{Q_C}{W_{on}} \leq \frac{1}{\frac{T_H}{T_C} - 1}$

Efficiency/COP can be thought of as
“what you get out” divided by
“what you put in.”



Example Problem: Engine/Heat Pump COP

Given a hot reservoir at a temperature of $T_h = 373K$
and a cold reservoir at a temperature of $T_c = 293K$,
calculate the maximum efficiency ϵ of an engine
and the maximum COP of a heat pump between the
two reservoirs.

Heat Engine Efficiency:

Heat Pump COP:

Example Problem: Engine/Heat Pump COP

Given a hot reservoir at a temperature of $T_h = 373K$
and a cold reservoir at a temperature of $T_c = 293K$,
calculate the maximum efficiency ϵ of an engine
and the maximum COP of a heat pump between the
two reservoirs.

Heat Engine Efficiency:

$$\epsilon = 1 - \frac{T_c}{T_h} \approx 0.21$$

Heat Pump COP:

Example Problem: Engine/Heat Pump COP

Given a hot reservoir at a temperature of $T_h = 373K$
and a cold reservoir at a temperature of $T_c = 293K$,
calculate the maximum efficiency ϵ of an engine
and the maximum COP of a heat pump between the
two reservoirs.

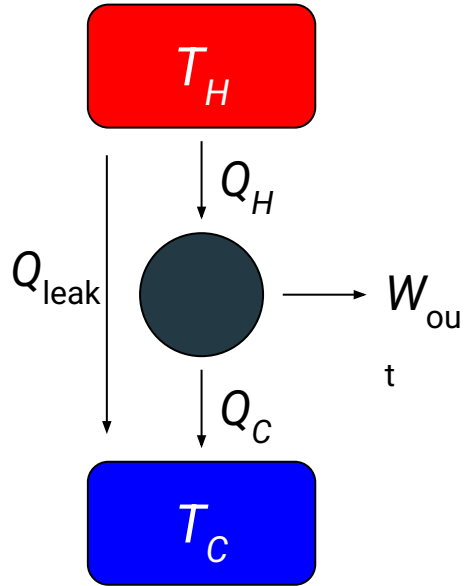
Heat Engine Efficiency:

$$\epsilon = 1 - \frac{T_c}{T_h} \approx 0.21$$

Heat Pump COP:

$$COP = \frac{1}{1 - T_c/T_h} \approx 4.7$$

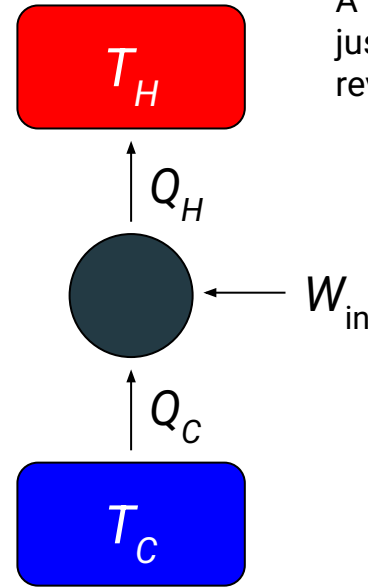
Engine, Pump, and Refrigerator Diagrams



Heat Engine Diagram

In a leaky engine, the heat we put in is $Q_H + Q_{leak}$, which affects the efficiency.

$$Q_H = W_{out} + Q_C$$
$$\epsilon = W_{out} / (Q_H + Q_{leak})$$
$$\epsilon_C = 1 - (T_C / T_H)$$



Heat Pump/Refrigerator Diagram

A heat pump or fridge is just a heat engine run in reverse.

$$Q_H = W_{in} + Q_C$$

Gibbs Free Energy

- Useful when temperature and pressure are fixed

$$G = U - T_{env}S + pV$$

- **Minimizing Gibbs of a system will maximize total (system + environment) Entropy**
 - As a system approaches equilibrium, free energy will decrease to a minimum
- **Fundamental Thermodynamic Relation in Equilibrium:**
 - $TdS = dU + pdV - \mu dN$
 - $\mu = (dG/dN) \rightarrow \mu N = G$ (at fixed temperature and pressure)
 - Equilibrium favors lowest μ

Good luck!

Feel free to ask any questions you may have.

You got this!

